

Calculations concerning the shaft



a)

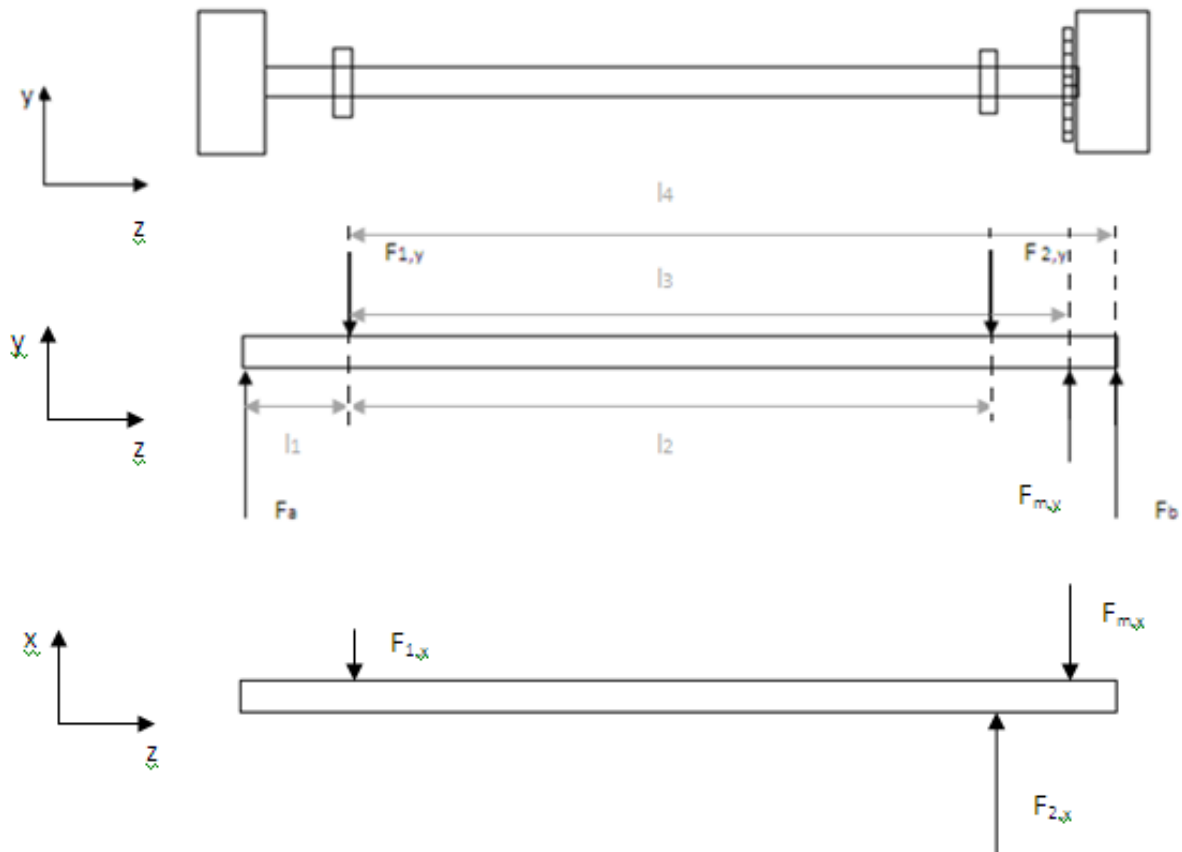


Figure: 2D-drawings of axle.

b)

- 2 x wheels
- 2 x Bearings
- 1 x Gear

c)

$$F_a = F_b = \frac{1}{2} m * g = \frac{1}{2} * 0.962 * 9.81 = 4.72 \text{ N}$$

$$\rightarrow \text{If } T_m = 5.09 * 10^{-3} \text{ Nm} \quad (T_m = \text{maximum torque of motor})$$

$$F_m = T_m / R_1 = (5.09 * 10^{-3}) / (3 * 10^{-3}) = 1.70 \text{ N}$$

$$\Rightarrow F_{m,x} = F_m * \cos(20^\circ) = 1.59 \text{ N}$$

$$\Rightarrow F_{m,y} = F_m * \sin(20^\circ) = 0.58 \text{ N}$$

Given: $l_1 = 28.5 \text{ mm}$

$$l_2 = 208 \text{ mm}$$

$$l_3 = 230 \text{ mm}$$

$$l_4 = 245 \text{ mm}$$

$$\underline{y:} \quad \left\{ \begin{array}{l} \sum F_y = 0 \\ F_{1,y} + F_{2,y} = F_a + F_b + F_{m,y} \\ \sum M_1 = 0 \\ -F_a * l_1 - F_{2,y} * l_2 + F_{m,y} * l_3 + F_b * l_4 = 0 \end{array} \right.$$
$$\Rightarrow F_{2,y} = F_{m,y} * l_3 - F_a * l_1 + F_b * l_4 = 5.55 \text{ N}$$
$$\Rightarrow F_{1,y} = F_a + F_b + F_{m,y} - F_2 = 4.47 \text{ N}$$

$$\underline{x:} \quad \left\{ \begin{array}{l} \sum F_x = 0 \\ F_{2,x} = F_{m,x} + F_{1,x} \\ \sum M_1 = 0 \\ F_{2,x} * l_2 - F_{m,x} * l_3 = 0 \end{array} \right.$$
$$\Rightarrow F_{2,x} = (F_{m,x} * l_3) / l_2 = 1.76 \text{ N}$$
$$\Rightarrow F_{1,x} = F_{2,x} - F_{m,x} = 0.16 \text{ N}$$

$$M_{b,max} = (M_{b,y}^2 + M_{b,x}^2)^{(1/2)} = (186.5^2 + 33.3^2)^{(1/2)} = 189.4 * 10^{-3} \text{ Nm}$$

$$T_{max} = T_t / 2 = (T_m / i) / 2 = (5.09 * 10^{-3} / 7.38) / 0.345 = 345 * 10^{-3} \text{ Nm} \quad (T_t = \text{Torque of gear})$$

Bending stress:

Given: $D = 6 \text{ mm}$

$D = 5.4 \text{ mm}$

$M_b = 189 * 10^{-3} \text{ Nm}$

$T_{\max} = 945 * 10^{-3} \text{ Nm}$

$$\sigma_b = M_b / w_b = (M_b * 32 * D) / [\pi * (D^4 - d^4)] = 25.9 \approx 26 \text{ MPa}$$

Torsion stress:

$$\tau_T = T / w_T = (T * y_{\max}) / I_p = [T * (D/2) * 16] / [\pi * (D^4 - d^4)] = 11.8 \approx 12 \text{ kPa}$$

Shear stress:

With: $y_z = (4\pi/3) * [(D/2) + (d/2)]$

$$= (4\pi/6) * (D + d)$$

$$A = \frac{1}{2} * \pi * [(D/2)^2 + (d/2)^2]$$

$$= (\pi/8)(D^2 + d^2)$$

$$I = w_b * y_{\max} = [\pi * (D^4 - d^4)] / 64$$

$$\tau = (M_b * S_y) / (I * b_m) = (M_b * y_z * A) / [(\pi * (D^4 - d^4)) * (D - d) / 64]$$

$$= [M_b * (4/6\pi) * (D + d) * (1/8) * (D^2 - d^2)] / [\pi * (D^4 - d^4) * (D - d) / 64]$$

$$= 29.8$$

$$\approx 30 \text{ kPa}$$

- d) In motion the shaft experiences some different forces.
 The resistance in the bearings is a reason for little differences.
 When the surface is not flat, the forces caused by the weight can peak to higher amplitudes.